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# 超临界二氧化碳与液态金属耦合换热特性研究进展

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**[摘要]** 液态金属快堆/太阳能光热系统与超临界二氧化碳(S-CO<sub>2</sub>)布雷顿循环发电系统深度融合, 必将引领能源动力领域革命性发展。由于液态金属与S-CO<sub>2</sub>的特殊物理性质, 液态金属普朗特数远低于常规流体, S-CO<sub>2</sub>的物理性质奇异性变化, 其流动换热特性与常规流体存在显著差别, 其流动与传热机理比较复杂, 耦合传热机理尚不明朗。本文归纳总结了国内外关于S-CO<sub>2</sub>、液态金属、耦合换热与耦合换热器在实验、数值模拟、传热预测模型的主要研究成果, 指出液态金属与S-CO<sub>2</sub>流动换热及其耦合传热研究中存在的问题, 为先进动力循环系统以及多工质耦合动力系统的设计和安全运行提供参考依据。

**[关键词]** 液态金属; 超临界二氧化碳; 流动换热; 铅冷快堆; 钠冷快堆; 太阳能光热

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## Research progress on conjugated heat transfer between supercritical carbon dioxide and liquid metals

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**Abstract:** Liquid metal fast reactor/concentration solar power system coupled with supercritical carbon dioxide (S-CO<sub>2</sub>) Brayton cycle power system will surely lead the revolutionary development in the field of energy and power in future. Due to the special properties of liquid metal and S-CO<sub>2</sub>, the *Pr* number of liquid metals is low; the physical properties of S-CO<sub>2</sub> steeply varies in the pseudocritical region, thus its flow and heat transfer characteristics are different from those of conventional fluids, their heat transfer mechanism is relatively complex, and the conjugated heat transfer mechanism is not clear. This study summarizes the main research results of supercritical CO<sub>2</sub>, liquid metal, conjugated heat transfer and conjugated heat exchanger at home and abroad in experiments, numerical simulation and prediction models, points out the problems in the research of CO<sub>2</sub>, liquid metals and their conjugating heat transfer between the two fluids, the discussion can provide valuable reference for the design and safe operation of advanced power cycle and multiple working fluids coupling power systems.

**Key words:** liquid metal; supercritical carbon dioxide; heat transfer; LFR; SFR; CSP

液态金属快中子反应堆是极具发展潜力的四代核反应堆型<sup>[1]</sup>, 具有运行安全性好、独特的核废

料嬗变和核燃料增殖能力。采用液态金属(liquid metal, LM)作为反应堆冷却剂具有高热焓、高功率

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密度和良好的热传递特性,非常适合核动力潜艇、航母等对小体积、高功率、高灵活性有特别需要的平台,也可满足其他多重环境中高效、灵活等复杂用电需求的平台(如可移动小型模块化电源、分布式电网供应等<sup>[2]</sup>),常采用液态金属钠(钠冷快堆, SFR)和液态铅/铅铋(铅冷快堆, LFR)作为冷却工质。同时,液态钠由于沸点高,可承受高温高效能源转换系统,使用钠作为传热介质的太阳能聚光发电系统(concentrated solar power, CSP)亦备受关注<sup>[3]</sup>。

兼具紧凑、高效的超临界二氧化碳(S-CO<sub>2</sub>)布雷顿循环发电系统因其循环特性与金属快堆/太阳能光热系统的小体积、高功率、高灵活性应用需求高度匹配,被认为是耦合先进新型快堆与先进太阳能发电的理想动力循环<sup>[4-5]</sup>。快堆/太阳能光热系统与S-CO<sub>2</sub>布雷顿循环发电系统深度融合,必将引领能源动力系统跨越式发展。

中间耦合式LM-S-CO<sub>2</sub>换热器(IHX)是液态金属反应堆、太阳能光热集热器与S-CO<sub>2</sub>布雷顿循环动力系统耦合的关键热运输设备,其性能的优劣直接影响发电系统效率、动态特性及安全性<sup>[6]</sup>。

IHX的传热效率将极大地影响整个系统的发电效率;液态金属与S-CO<sub>2</sub>的流动与传热特性对于反应堆设计及安全运行具有重要意义<sup>[7]</sup>。由于液态金属与S-CO<sub>2</sub>的特殊性质,其流动传热现象比较复杂,流动传热机理还不明朗。本文将归纳总结国内外S-CO<sub>2</sub>、液态金属传热特性及不同工质间耦合换热的主要研究成果,指出前人在液态金属与S-CO<sub>2</sub>流动换热研究中存在的不足,为先进动力循环系统以及多工质耦合动力系统设计提供参考依据。

## 1 S-CO<sub>2</sub>与液态金属物理性质

S-CO<sub>2</sub>在拟临界区内呈奇异性变化,热物理性质剧烈变化。图1为不同温度下多种液态金属和S-CO<sub>2</sub>的普朗特数对比(数据源于NIST Refprop)。

由图1可知,常规介质如空气的普朗特数约为0.8,而S-CO<sub>2</sub>的普朗特数在拟临界区内峰值达到几十,液态金属的普朗特数却在10<sup>-3</sup>~10<sup>-2</sup>量级。液态金属和S-CO<sub>2</sub>的普朗特数随温度变化趋势差异显著,液态金属的普朗特数远小于S-CO<sub>2</sub>。OCED/NEA出版的热物性手册<sup>[8]</sup>中总结了诸多学者对金属物性的研究结果,发现除温度以外,压力和环境湿度等因素对液态金属的热物性影响不大,因此将液态金属的热物性表示为温度的函数<sup>[8]</sup>。

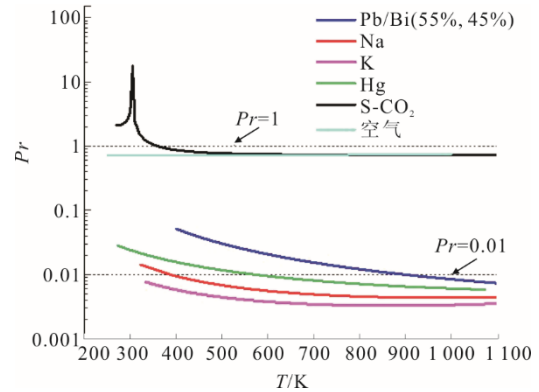


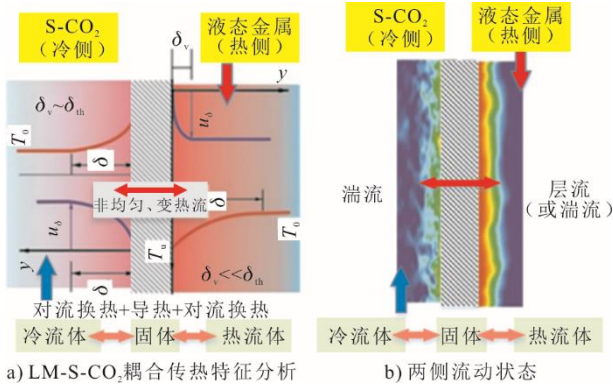
图1 液态金属与S-CO<sub>2</sub>普朗特数的变化  
Fig.1 Variation of Prandtl in liquid metal and S-CO<sub>2</sub>

## 2 S-CO<sub>2</sub>与液态金属耦合传热机理

液态金属的普朗特数和S-CO<sub>2</sub>相差几十倍,其湍流换热特性和普通流体亦存在显著差别,特别是在边界层内。对 $Pr \sim 1$ 的流体(如空气),速度边界层和温度边界层厚度相当,对流和热扩散能力相近;而对于 $Pr \ll 1$ 的液态金属,其温度边界层厚度远高于速度边界层(图2)<sup>[9]</sup>,温度梯度仅小部分在黏性底层中,传热几乎不受黏度影响。液态金属的高导热系数可以使热量快速传入流体核心区。液态金属的热扩散性远高于动量扩散能力,壁面温度分布更平滑,热传导机制占比增大,甚至处于主导地位。拟临界区内存在尖峰S-CO<sub>2</sub>,其湍流传热机制受黏性层作用显著,沿径向和轴向的湍流脉动呈现强非线性、非平衡特征,浮力效应、加速效应、累积效应增强。在大温差、高热流边界作用时,极易诱发异常传热。这是区别普通流体的重要特性。

在金属快堆与太阳能光热系统主换热器内,金属温度高达450~800℃,而S-CO<sub>2</sub>入口温度为250~600℃,冷、热流体的换热过程如图2a)所示,冷、热流体间换热端差高达200~300℃,通道两侧工质的温度不断变化,形成2种特殊介质(LM-S-CO<sub>2</sub>)在一种大温差、高变热流边界条件下的特殊耦合传热过程。

由于液态金属黏度较高,速度较小,热侧LBE流动状态可能处于层流或过渡流状态,而冷侧的S-CO<sub>2</sub>往往处于旺盛湍流状态,如图2b)所示,二者的流动状态差异显著。LM-S-CO<sub>2</sub>之间能否安全高效地进行耦合传热将极大地影响整个系统性能,而目前对于LM-S-CO<sub>2</sub>这2种特殊工质间耦合传热规律和计算方法的研究均极为匮乏,耦合传热机理的认识不足以及合理计算模型的缺乏,为高效换热器的设计与研发带来困难。

图2 LM-S-CO<sub>2</sub>耦合传热过程Fig. 2 Conjugated heat transfer process in LM-S-CO<sub>2</sub>

### 3 S-CO<sub>2</sub> 传热特性研究

超临界工质在实际工程中已得到广泛应用,从20世纪50年代起,国内外学者已开展多种超临界工质的实验和理论研究,其中包括水、二氧化碳、液氮、液氦、液氢和多组分烃类等。超临界流体虽然没有亚临界压力下的相变过程,但在拟临界区,超临界流体物性奇异性变化引发一系列新的问题,如管内传热模式的改变及工质传热性能的变化<sup>[10]</sup>。

#### 3.1 S-CO<sub>2</sub> 传热特性实验研究

超临界流体对流传热过程已开展了大量实验研究<sup>[11]</sup>,对其流动与传热规律形成一定认识<sup>[12-14]</sup>,积累了丰富的超临界流体流动与传热实验数据。各研究者分别尝试采用不同形式的物性比、浮升力修正因子和热加速修正因子对超临界流体的传热系数计算关联式进行不同程度修正,以期达到提高预测精度的目的。目前见诸于公开文献中的类似传热系数计算关联式达40个之多;然而,不同研究者提出的传热系数计算关联式表现形式迥异,而且各个关联式均是根据实验数据拟合,受各自具体实验条件的限制,只适用于部分工况,超出适用范围其误差会增大。这些研究针对不同流动参数、结构参数下超临界流体的传热特性开展的大量研究,探讨了热流密度、压力、质量流率和管径<sup>[15-17]</sup>等参数对传热的影响<sup>[18-19]</sup>。但对于超高参数条件下的传热数据仍极为稀少<sup>[20-21]</sup>,现有数据无法涵盖换热器实际温压窗口内的传热特性。

#### 3.2 S-CO<sub>2</sub> 数值模拟分析

超临界流体湍流流动具有随机性、耗散性、有旋性等特点,很难通过理论求解获得湍流流动信息,实验方法耗资巨大且难以获得局部流场信息,目前很多学者采用数值模拟方法对S-CO<sub>2</sub>湍流特性进行研究。但受热物理性质、浮升力、热加速效应

以及上游累积效应的影响,工质的输运规律与热质传递现象异常复杂,目前利用各种湍流模型获得的数值模拟结果也存在较大差异。学术界尝试选取多种形式的各向同性、各向异性湍流模型以及更复杂的雷诺应力模型进行计算研究(如Low-Re、 $k-\epsilon$ 、 $k-\omega$ 、RNG、SST、RSM等),以期挑选出最可靠的可用于超临界流体的流动与传热湍流模型,遗憾的是不同学者得到的结论迥异,至于具体哪种湍流模型是最可靠的模型,依然没有定论。Tang<sup>[22]</sup>、Mohseni<sup>[23]</sup>和Bae<sup>[24]</sup>等人通过采用可变湍流普朗特数和更先进的浮力产生湍流动能的计算方案<sup>[24]</sup>,或在低雷诺数模型中更适当地处理响应局部变化(特别是近壁面处)的阻尼函数<sup>[25]</sup>,构建高阶湍流模型等方法来提升计算结果的精度,取得了良好的效果<sup>[26-28]</sup>,但这些模型同样存在只适用于部分工况的问题,对于强烈非线性物性畸变区的传热预测仍存在较大误差,仍有待进一步研究。

#### 3.3 S-CO<sub>2</sub> 传热预测关联式

研究者们结合理论分别针对不同工况条件下进行经验拟合、预测-矫正技术等提出了传热特性预测关联式,部分关联式见表1。

采用表1中的预测关联式与Bae<sup>[33]</sup>的实验值进行了比较,结果如图3所示。研究发现Mokry(2011)关联式的预测精度最高,各项指标均优于现有其他关联式。对于中、高质量流速条件下S-CO<sub>2</sub>传热恶化的预测,推荐采用Mokry(2011)关联式和Jackson(2008)关联式,而低质量流速条件下S-CO<sub>2</sub>传热推荐采用Zhang关联式。

显然,各预测关联式的适用范围仍较窄且使用较为复杂,后续应继续拓宽模型参数的适用性,建立高普适性S-CO<sub>2</sub>传热预测模型。同时,已有计算关联式对高参数条件下小通道内的S-CO<sub>2</sub>传热预测的适用程度也需进一步检验。

## 4 液态金属流动与换热特性研究

#### 4.1 液态金属流动与换热特性实验研究

按照比拟理论,由于液态金属性质具有相似性(如液态铅铋、钠、钠钾、汞等),因此液态金属将具有相似流动换热特征,其经验关联式理应具有一定的普适性。然而,通过对早期开展的大量液态金属实验研究(主要集中于俄罗斯、美国以及欧盟)以及换热经验关系式的总结发现,不同液态金属间的实验结果存在明显差异。

表 1 S-CO<sub>2</sub> 传热预测关联式  
Tab. 1 Heat transfer correlations for S-CO<sub>2</sub>

文献	关联式
Fewster(1976)	$Nu_b = 0.018 3Re_b^{0.82} Pr_b^{0.5} \left(\frac{\rho_w}{\rho_b}\right)^{0.3} \left(\frac{c_p}{c_{p,b}}\right)^n$
Kim J(2007)	$Nu_b = 0.023Re_b^{0.8} Pr_b^{0.4} \left(\frac{\xi_M}{\xi_F}\right) \left(\frac{\rho_w}{\rho_b}\right)^n \left(\frac{c_p}{c_{p,b}}\right)^{0.6}$
Kim H(2008)	$Nu_b = 0.018 2Re_b^{0.824} Pr_b^{-0.515} \left(\frac{\rho_w}{\rho_b}\right)^{0.299}$
Jackson(2008)	$Nu_b = 0.022Re_b^{0.8} Pr_b^{0.4} \left(\frac{\rho_w}{\rho_b}\right)^{0.3} \left(\frac{c_p}{c_{p,b}}\right)^n$
Pioro(2012)	$Nu_w = 0.003 8Re_w^{0.96} Pr_w^{-0.14} \left(\frac{\rho_w}{\rho_b}\right)^{0.84} \left(\frac{\lambda_w}{\lambda_b}\right)^{-0.75} \left(\frac{\mu_w}{\mu_b}\right)^{-0.22}$
reda(2012)	$Nu_w = 0.001 5Re_w^{1.03} Pr_w^{-0.76} \left(\frac{\rho_w}{\rho_b}\right)^{0.46} \left(\frac{\lambda_w}{\lambda_b}\right)^{-0.43} \left(\frac{\mu_w}{\mu_b}\right)^{0.53}$
Mokry(2011)	$Nu_b = 0.006 1Re_b^{0.904} Pr_b^{0.684} \left(\frac{\rho_w}{\rho_b}\right)^{0.564}$
ang (2009) <sup>[29]</sup>	$Nu_b = 0.013 78Re_b^{0.907 8} Pr_b^{-0.617 1} \left(\frac{\rho_w}{\rho_b}\right)^{0.435 6} (Gr^*)^{-0.012} (q^*)^{0.060 5}$
Jackson(1979)	$Nu_b = 0.018 3Re_b^{0.82} Pr_b^{0.5} \left(\frac{\rho_w}{\rho_b}\right)^{0.3} \times \left[ 1 + \frac{10 000 Gr_b}{Re_b^{2.7} Pr_b^{0.5}} \right]^{-0.46}$
Bae(2009)	$Nu_0 = 0.021Re_b^{0.82} Pr_b^{0.5} \left(\frac{\rho_w}{\rho_b}\right)^{0.3} \left(\frac{c_p}{c_{p,b}}\right)^n$
Kim D(2010)	$Nu_b = 0.226Re_b^{1.174} Pr_b^{1.057} \left(\frac{\rho_w}{\rho_b}\right)^{0.571} \left(\frac{c_p}{c_{p,b}}\right)^{1.032} Ac^{0.489} Bu^{0.002 1}; Ac = \frac{q^+}{Re_b^{0.625} \left(\frac{\mu_w}{\mu_b}\right) \left(\frac{\rho_b}{\rho_w}\right)^{0.5}}; Bu = \frac{Gr^*}{Re_b^{3.425} Pr_b^{0.8} \left(\frac{\mu_w}{\mu_b}\right) \left(\frac{\rho_b}{\rho_w}\right)^{0.5}}$
Kim D-1(2011)	$Nu_b = 2.051 4Re_b^{0.928} Pr_b^{0.742} \left(\frac{\rho_w}{\rho_b}\right)^{1.305} \left(\frac{\mu_w}{\mu_b}\right)^{-0.669} \left(\frac{c_p}{c_{p,b}}\right)^{0.888} (q^*)^{0.792}$
Yang S(2005)	$Nu_b = 3.444 5(Nu_0)^{1.064 9} \left(\frac{\mu_b}{\mu_w}\right)^{-1.436 2} \left(\frac{c_p}{c_{p,b}}\right)^{0.977 1} \left(\frac{\lambda_b}{\lambda_w}\right)^{0.387 9} \left(\frac{T_b}{T_c}\right)^{1.973 7} \left(\frac{P}{P_c}\right)^{-0.449 3} \left(\frac{10 000q}{Gh_b}\right)^{0.253 3}$
Zhang Q <sup>[30]</sup>	$Nu_b = 1.4 \cdot Re_b^{0.020 4} Pr_b^{-0.012 7} \left(\frac{\mu_w}{\mu_b}\right)^{-0.694 7} \left(\frac{\rho_w}{\rho_b}\right)^{0.622} \left(\frac{c_p}{c_{p,b}}\right)^{0.955 4} Bu_{new}^{0.873 8}$
刘生晖等 <sup>[31]</sup>	$Nu_b = 0.001 47Re_b^{1.16} Pr_b^{0.62} \exp(4.65 \times 10^8 Bu^{2.86}) \times Ac^{0.31} \left(\frac{\mu_w}{\mu_b}\right)^{-0.31} \left(\frac{c_p}{c_{p,b}}\right)^{0.14} \left(\frac{k_w}{k_b}\right)^{-0.40} \times \left(\frac{\rho_w}{\rho_b}\right)^{7.49} \left[ 1 + \frac{2.23}{(L/D)^{0.211}} \right]$
杨温等 <sup>[32]</sup>	$Nu_b = 66Re_b^{1.098} Pr_b^{2.414} \left(\frac{\rho_w}{\rho_b}\right)^{2.751} \left(\frac{c_p}{c_{p,b}}\right)^{0.882} (q^*)^{1.185} (c_1)^{5.468}$

具有代表性的液态金属实验平台包括中国实验快堆（CEFR）钠快中子增殖反应堆、韩国 STELLA-1 大尺寸钠实验台<sup>[34]</sup>、瑞典皇家理工学院的 TALL 铅铋换热实验平台<sup>[35]</sup>、德国卡尔斯鲁厄研究中心的 KALLA 系列铅铋实验平台<sup>[36-37]</sup>、意大利 CIRCE 实验台<sup>[38]</sup>、印度原子研究中心 KTL 实验台<sup>[39]</sup>以及中国科学院核能安全技术研究所的 KYLIN 系列铅铋实验平台<sup>[40]</sup>。

巫英伟等全面综述了液态金属钠单相<sup>[41]</sup>与两相流动与传热特性<sup>[42]</sup>。张弘喆<sup>[43]</sup>研究了热管中冷却

水参数对钠钾合金热管传热性能。Ma 等人<sup>[44]</sup>研究液态铅铋合金（LBE）的自然循环能力以及反应堆热工水力学瞬态安全性。Lefhalm 等<sup>[45]</sup>、Marocco 等<sup>[36]</sup>分别进行了水平加热环管和垂直环管内 LBE 强迫对流换热实验，研究了强迫对流和浮力辅助流的情况。吕科锋等<sup>[40]</sup>开展了液态 LBE 在带绕丝棒束组件内的流动阻力特性和换热特性的实验研究。Xi 等人<sup>[46]</sup>对铅铋-氦气的实验回路的阻力特性和换热特性进行了实验研究。朱锋杰等<sup>[47]</sup>对环管内液态铅铋的流动换热特性进行了实验研究，指出导热在

对流换热中占主导地位, 液态 LBE 流场和温度场受到浮力显著影响。Marocco<sup>[36]</sup>提出了一个无量纲数

$c^* = Ra^{0.333}/Re^{0.8}Pr^{0.4}$ , 将浮升力和惯性力关联, 衡量强迫对流到混合和自然对流的转变。

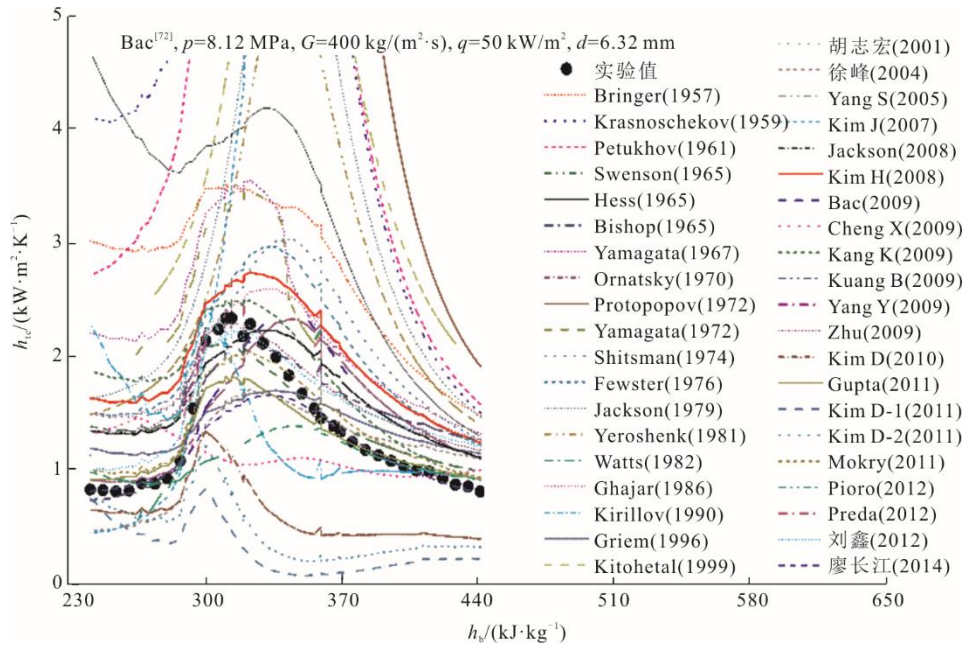


图 3 传热关联式预测值与 S-CO<sub>2</sub> 典型传热工况实验值比较

Fig. 3 Comparison of heat transfer correlations predicted value and S-CO<sub>2</sub> classic experimental data

表 2 给出了不同几何形状下 (含圆管、环管、三角形棒束、方形棒束、平板、矩形) 的换热关联式。由表 2 可以看出, 不同几何条件下液态金属换热计算关联式种类较多, 缺乏通用性。

为了解同一管型内传热关联式间的差异, 将表 2 中圆管的传热关联式进行了比较, 圆管  $Pr=0.02$ , 环管  $D/d=1.3$ , 结果如图 4 所示。

由图 4 可知, 各关联式的适用范围存在一定差别, 关联式预测的  $Nu$  差别甚至高达 1 倍之多。以 NaK、Hg、Na 为介质的实验数据以及关联式计算得到的传热系数总体高于液态铅铋的预测值, 表明其他液态重金属的传热关联式并不适用于预测液态钠、铅铋合金的传热性能。

综合比较实验数据与各关联式的预测值, 发现 Cheng & Tak 关联式对圆管内液态铅铋合金的传热预测效果最佳, 而 Ching 关联式对于 NaK 的预测效果最优, 因而圆管内液态铅铋合金换热关联式推荐 Cheng & Tak 关联式; 相较而言, 环管内液态铅铋换热数据较少, 各关联式计算得到的努塞尔数差异较小, 为获得较为保守的结果, 建议使用 Jäger 关联式对环管内传热性能进行预测。

各关联式计算结果间出现显著偏差, 一部分归结为早期的研究中, 部分研究者未明确给出液态金属的具体名称, 也未指出关联式对于其他液态金属

的适用程度<sup>[54]</sup>。再者, 介质物理性质的变化可能是导致偏差较大的原因<sup>[55]</sup>。经调查: 液态金属氧化物在壁面沉积产生的壁面接触热阻, 导致计算得到的壁面温度较低<sup>[66]</sup>; 局部因素导致该处的液态金属流体与壁面脱离, 壁面润湿状态不同, 未完全润湿状态会削弱传热<sup>[67]</sup>; 液态金属密度分布不均, 不同浮力条件对温度场和速度场分布造成不同影响<sup>[68]</sup>; 流道材料被腐蚀、脆化、疲劳等引发的传热差异<sup>[69]</sup>。

综上所述, 目前针对液态金属的实验研究还比较有限, 现有的大部分实验是针对其传热性能、结构材料性能、化学控制、自然循环以及加热棒束的强迫对流换热等进行研究, 对其流动传热研究存在薄弱之处。实验主要集中在大管径、简单几何形状的研究。早期得到的大量实验和关系式, 大多是针对汞和钠钾合金, 实验参数、边界条件等存在缺失, 难以验证, 针对 LM 流动换热性能的实验研究仍然较少。换热经验关联式不具有统一性、普遍性, 对于液态金属流动换热特性的实验研究仍要加强。

#### 4.2 液态铅铋合金流动传热特性数值模拟研究

直接数值模拟 (DNS) 方法直接对控制方程进行求解, 能提供十分准确而全面的数据, 对于研究液态金属在低贝克来数时湍流换热机理十分有效。因此, 越来越多的学者尝试采用 DNS 对 LM 流动换热进行计算分析。采用 DNS 方法也面临着计算资

源需求量过高的困扰,因而目前使用 DNS 方法对液态金属流体的流动换热研究局限于较小的雷诺数 ( $Re < 5\,500$ )、简单几何形状如圆管、平板间流动

换热情况。大涡模拟 (LES) 在保证计算结果一定精度的同时大大减少计算量,部分学者尝试采用 LES 对液态金属流动换热进行计算分析。

表 2 不同几何液态金属传热预测关联式  
Tab. 2 Heat transfer correlations of different geometry liquid metals

形状	作者	传热预测关联式	介质	适用范围
圆管	Skupinski <sup>[48]</sup>	$Nu=4.82+0.018\,5Pe^{0.827}$	NaK	$0 \leq Pr \leq 0.1, 10^4 \leq Re \leq 5 \times 10^5$ $58 \leq Pr \leq 1.31 \times 10^4$
	Lyon <sup>[49]</sup>	$Nu=7.0+0.025Pe^{0.8}$		$0 \leq Pr \leq 0.1, 0 < Pe < 1.5 \times 10^4$
	Sleicher <sup>[50]</sup>	$Nu=6.3+0.016\,7Pe^{0.85}Pr^{0.88}$	NaK	$0.004 < Pr < 0.1, 10^4 < Re < 10^6$
	Chin <sup>[51]</sup>	$Nu=5.6+0.016\,5Pe^{0.85}Pr^{0.01}$	Na, NaK	$0 < Pr < 0.1, 10^4 < Re < 5 \times 10^6$
	Stromquist <sup>[52]</sup>	$Nu=3.6+0.018Pe^{0.8}$	Hg	$88 < Pe < 4\,000$
	Ibragimov <sup>[53]</sup>	$Nu=4.5+0.014Pe^{0.8}$	LBE	$0.02 \leq Pr \leq 0.027, 8 \times 10^3 \leq Re \leq 4 \times 10^5$ $200 \leq Pe \leq 10^4$
环管	Cheng & Tak <sup>[54]</sup>	$Nu=A+0.018Pe^{0.8}$	LBE	$A = \begin{cases} 4.5 & Pe \leq 1\,000 \\ 5.4 - 9 \times 10^{-4} Pe & 1\,000 < Pe < 2\,000 \\ 3.6 & Pe \geq 2\,000 \end{cases}$
	Jaeger <sup>[55]</sup>	$Nu=(D/d)^{0.3} \cdot [4.75+0.017\,5Pe^{0.8}]$		$D/d \leq 7, Pe < 10\,000$
	朱锋杰 <sup>[56]</sup>	$Nu=(4.05+0.021Pe^{0.8})(d_2/d_1)^{0.3}$	LBE	$100 < Pe < 450$
	仇子铖 <sup>[57]</sup>	$Nu=(4.37+0.016\,8Pe^{0.8})(d_2/d_1)^{0.3}$	Na	
	Seban <sup>[58]</sup>	$Nu = \begin{cases} 5.8 + 0.020Pe^{0.8} & \frac{R_o}{R_i} \leq 1.4 \\ 5.25 + 0.018\,8Pe^{0.8} \left(\frac{R_o}{R_i}\right)^{0.3} & \frac{R_o}{R_i} > 1.4 \end{cases}$		
三角形棒束	Ushakov <sup>[59]</sup>	$Nu=7.55x-20x^{-1.3}+0.041x^{-2}Pe^{0.56+0.19x}$		$0 \leq Pe \leq 4\,000, 1.3 \leq x \leq 2.0$
	Borishanskii <sup>[60]</sup>	$Nu_1=24.15lg(-8.12+12.76x-3.65x^2)$ $Nu_2=Nu_1+0.017\,4[1-e^{-6(x-1)}](Pe-200)^{0.9}$	Na	$60 \leq Pe \leq 200, 1.1 \leq x \leq 1.5$ $200 \leq Pe \leq 2\,200, 1.1 \leq x \leq 1.5$
	Graeber <sup>[61]</sup>	$Nu=0.25+6.2x+(0.032x-0.007)Pe^{0.8-0.024x}$		$110 \leq Pe \leq 4\,300, 1.25 \leq x \leq 1.95$
	Mikityuk <sup>[62]</sup>	$Nu=0.047(1-e^{-3.8(x-1)})(Pe^{0.77}+250)$	NaK, Hg	$30 \leq Pe \leq 5\,000, 1.10 \leq x \leq 1.95$
方形棒束	Subbotin <sup>[63]</sup>	$Nu = 0.58 \left(\frac{4x^2}{\pi} - 1\right)^{0.55} Pe^{0.45}$		$80 \leq Pe \leq 4\,000, 1.1 \leq x < 1.5$
	Zhukov <sup>[64]</sup>	$Nu=7.55x-14x^{-5}+0.007Pe^{0.64+0.246x}$	Pb	$60 \leq Pe \leq 2\,000, 1.25 < x < 1.46$
	Mikityuk <sup>[62]</sup>	$Nu=0.047(1-e^{-3.8(x-1)})(Pe^{0.77}+250)$	NaK, Hg	$30 \leq Pe \leq 4\,000, 1.10 < x < 1.95$
平板	Jäger <sup>[65]</sup>	$Nu=5.268\,6+0.001\,04 \cdot (Re \cdot Pr)^{1.171}$		长/宽 $\geq 10$ $Pr \cdot Re < 3\,000$
矩形	Jäger <sup>[65]</sup>	$Nu=7.833+0.013\,8 \cdot (Re \cdot Pr)^{0.8}$		长/宽 $< 10$ $Pr \cdot Re < 3\,000$

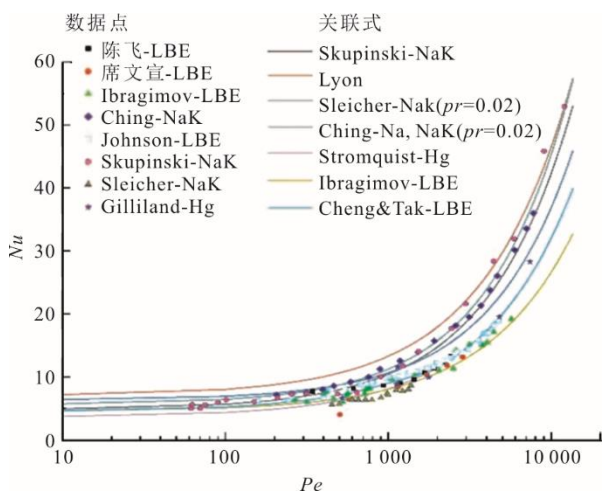


图 4 圆管内换热关联式的比较

Fig. 4 Comparison of different heat transfer correlations in circular channels

Grötzbach 和 Wörner 等人<sup>[70]</sup>通过 DNS 和 LES 对液态重金属的局部传热特性进行了研究,对湍流涡得到很好的捕捉。Kawamura<sup>[71]</sup>通过 DNS 对不同雷诺数下液态铅铋平板流动换热进行研究,发现对于比较低普朗特数流体 ( $< 0.2$ ),  $Pr_i$  分布不再是个常数,远大于 0.9。Bricteux 等人<sup>[72]</sup>使用 LES 和 DNS 方法对低雷诺数下液态金属在两平板间的流动特性进行数值模拟,指出将涡扩散的概念应用到湍流热传导,是不适合低普朗特数液态重金属流体的,对于液态 LBE 其湍流普朗特数的概念必须谨慎使用。Redjem-Saad 等<sup>[73]</sup>使用 DNS 方法对不同普朗特数流体在圆管内的流动传热进行模拟,比较液态金属和其他高普朗特数流体的流动传热特性。Duponcjeel 等人<sup>[74]</sup>通过 LES 方法研究液态金属在高雷诺数、平

板间流动中的近壁面温度分布特性，评估了其他湍流普朗特数模型，Kays 模型表现最好。席文宣等<sup>[75]</sup>运用 LES 方法，选取 Cheng&Tak 的  $Pr_t$  模型，对恒热流圆管内 LBE 强制对流换热特性开展了数值计算。祝家银<sup>[76]</sup>采用 LES 模拟分析了较高雷诺数、不同浮升力下 LBE 的湍流换热过程，获得了各浮升力下圆管内的湍流统计。吕逸君<sup>[77]</sup>采用 LES 研究了液态铅铋在环管中的流动特性，并采用 LES 的计算结果对雷诺时均方法（RANS 方法）进行评估，发现采用  $Pr_t=1$  和 Kays 模型均有较大误差，采用  $Pr_t=3.2$  所得结果最好。Ould-Rouiss 等人<sup>[78]</sup>采用 DNS 发现普朗特数的增加强化了环管换热，且分析了普朗特数对温度场、湍流热通量等参数的影响。Zhao 等人<sup>[79]</sup>则利用 DNS 方法研究了不同浮力条件下金属的对流换热，分析浮力作用对  $Nu$ 、摩擦系数的影响。然而实际工程应用比较复杂，单纯依靠 DNS 和 LES 模拟是不现实的，还是要依托于经济性更好的 RANS，因此需要开展大量湍流模型的修正和开发工作。

寻求并开发准确、可靠的适用于低普朗特数液态金属的 RANS 模型仍是获得准确可靠液态金属换热预测性能的关键点，而现有的液态金属湍流普朗特数模型在适用性方面仍缺乏一致性。RANS 通过引入湍流普朗特数  $Pr_t$ （该概念源于雷诺比拟）将湍流热扩散与湍流动量扩散建立联系，其在整个控制域内经常设为 0.85 的常数，非常适用于常规流体。而对于液态金属，其普朗特数较常规流体低约 2 个数量级，雷诺比拟理论失效。Thiele<sup>[80]</sup>、Grötzbach<sup>[81]</sup>、Cheng<sup>[54]</sup>、Ma<sup>[82]</sup>等人采用 RANS 方法中的不同湍流模型，对液态金属流动传热特性进行研究。研究表明，不同湍流模型下获得的结果各不相同，并且与实验结果对比存在较大差异。因此，部分学者提出改进湍流普朗特数模型来提高 RANS 方法对液态金属流动换热模拟的准确性，改进后湍流普朗特数  $Pr_t$  模型见表 3。

为校验不同  $Pr_t$  模型的适用程度，将各模型进行比较与分析。由表 3 可知，除 Kays 模型以外，其余湍流普朗特数模型与无量纲的贝克来数  $Pe$ 、雷诺数  $Re$ 、普朗特数  $Pr$  有关。图 5 比较了上述  $Pr_t$  模型。从图 5 可以看出，Reynolds 模型的  $Pr_t$  随  $Pe$  变化最不明显，Kays 模型的  $Pr_t$  明显小于其他模型，Cheng & Tak 提出的  $Pr_t$  模型最大。当  $Pe$  无限大时，Jischa 模型趋近于 0.90，Kays 模型趋近 0.85。

通过修改  $Pr_t$  模型提升低普朗特数流体湍流换热的准确性，取得了一定成效。表 4 对 RANS 方法

中  $Pr_t$  模型的适用性进行了总结。显然，各湍流普朗特数模型的适用性仍存在较大差异，缺乏统一。根据目前学者的研究比较，较多推荐 Kays 模型和 Cheng&Tak 模型。

表 3 湍流普朗特数模型  
Tab. 3 Turbulent Prandtl number model

年份	作者	湍流普朗特数模型
1963	Aoki	$Pr_t^{-1} = 0.014Re^{0.45}Pr^{0.2} \left[ 1 - \exp\left(-\frac{1}{0.014Re^{0.45}Pr^{0.2}}\right) \right]$
1975	Reynolds	$Pr_t = (1 + 100Pe^{-0.5}) \left( \frac{1}{1 + 120Re^{-0.5}} - 0.15 \right)$
1979	Jischa	$Pr_t = 0.9 + \frac{182.4}{PrRe^{0.888}}$
1994	Kays	$Pr_t = 0.85 + \frac{0.7}{Pr \frac{\nu_t}{\nu}}$ ( $\nu_t$ 为涡黏性系数)
2006	Cheng&Tak	$Pr_t = \begin{cases} 4.12 & Pe \leq 1\ 000 \\ \frac{0.01Pe}{[0.018Pe^{0.8} - (7.0 - A)]^{1.25}} & 1\ 000 < Pe \leq 6\ 000 \end{cases}$
		$A = \begin{cases} 5.4 - 9 \times 10^{-4} Pe & 1\ 000 < Pe \leq 2\ 000 \\ 3.6 & 2\ 000 < Pe \leq 6\ 000 \end{cases}$

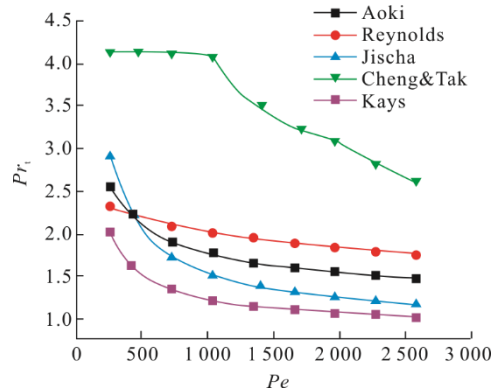


图 5 液态金属的湍流普朗特数模型比较  
Fig. 5 Comparison of turbulent Prandtl models in LM

表 4 RANS 方法中  $Pr_t$  模型的适用性  
Tab. 4 The applicability of  $Pr_t$  models in RANS

作者	适配条件	推荐湍流普朗特数模型
陈飞等 <sup>[83]</sup>	恒热流边界	Cheng&Tak 提出的 $Pr_t$ 模型
	恒壁温边界	Jischa 提出的 $Pr_t$ 模型
Marocco 等 <sup>[84]</sup>	圆管和环管	Kays 提出的 $Pr_t$ 模型
吴闽强等 <sup>[85]</sup>	半周加热	Cheng 提出的 $Pr_t$ 模型
葛志浩等 <sup>[86]</sup>	棒束子通道	Kays 和 Aoki 提出的 $Pr_t$ 数模型, $Pr_t=1.5$
Duponcjeel <sup>[74]</sup>	平板	Kays 提出的 $Pr_t$ 模型
吕逸君 <sup>[77]</sup>	环管	$Pr_t=3.2$
Lei <sup>[87]</sup>	圆形	$Pr_t=0.85+2.5/Pe_t$

## 5 LM-S-CO<sub>2</sub> 耦合换热与耦合换热器

耦合对流换热过程广泛存在于各类换热器中。耦合传热实验通常以获得总传热系数  $K$  为目标，通过计算两侧冷、热流体换热系数的方法来确定，因

此冷、热侧介质换热系数的准确测定和预测模型是耦合换热计算方法构建的关键。Pitla 等人<sup>[88]</sup>实验测量了不同温区内 S-CO<sub>2</sub> 与水在多段套管内的耦合换热特性。文献[89-90]中采用类似的方法分析了气体冷却器内传热系数。显然,相较于常规换热器测量方法,分段式测量方法可提供更为准确的测量结果。Kang 等人<sup>[91]</sup>研究了钠-空气耦合换热器中顺排与叉排管束时换热与阻力。Khalesi 等人<sup>[3]</sup>研究了 S-CO<sub>2</sub> 和钠在微通道内耦合换热规律。

近年来,有关 S-CO<sub>2</sub> 及其在微孔道换热器(PCHE 为最典型的微孔道换热器之一)内不同形状通道的流动与换热特性开展了大量实验与计算分析,如水平半圆直通道的<sup>[92]</sup>、Zigzag 型流道<sup>[93]</sup>、波浪型流道<sup>[94]</sup>、非连续的 S 型鳍片<sup>[95]</sup>、波浪型通道<sup>[96]</sup>、机翼形<sup>[97]</sup>。部分学者对各类不同形状微小通道内的流动与换热特性进行了细致比较,发现在相同换热量的情况下,Zigzag 型流道的 PCHE 换热器芯体尺寸远小于半圆直通道的 PCHE 换热器,且拥有更为优秀的换热性能<sup>[98]</sup>。Nobuyoshi Tsuzuki 等<sup>[95]</sup>设计了一种非连续的 S 型鳍片流道,可将换热器压降降低至 Zigzag 型的 1/5。文献[99]比较了三角形、正弦曲线型、梯形结构通道换热性能,研究发现梯形结构拥有最优的换热性能,但阻力最大。文献[100]比较了 Z 形通道、弧形通道、梯形通道、直通道的流动传热特性,研究发现:微通道横截面积相等时,Z 形通道传热系数最高,弧形通道次之;Z 形通道压损最高,直通道的压损最低。

值得指出的是,现有传热实验大都以锅炉水冷却壁、蒸汽发生器传热为背景,采用恒定热流的加热边界条件进行研究,缺乏 LM-S-CO<sub>2</sub> 工作介质环境相似条件下的耦合传热实验研究。在 LM-S-CO<sub>2</sub> 主换热器中,大温差、高变热流的加热条件使近壁面流体产生局部非均匀的浮升力效应和热加速作用,在附加力的作用下,局部的常规状态下的热质平衡被打破,热阻分布和传热效率呈现不均匀状态,导致 S-CO<sub>2</sub> 的湍流特征及耦合传热过程更加复杂。然而,超临界流体在径向/轴向的非线性物性畸变与传热之间存在何种关联,特别是在大温差、高变热流边界条件下,这种畸变规律如何与耦合传热特性建立联系目前尚未见深入的分析。

## 6 结 论

液态金属快堆与液态金属太阳能光热系统耦合紧凑、高效、灵活的 S-CO<sub>2</sub> 布雷顿循环发电系统,具有革命性的应用潜力。本文全面总结了 S-CO<sub>2</sub>、

液态金属以及二者间的耦合传热特性的研究成果,分析了液态金属与 S-CO<sub>2</sub> 流动换热及其耦合传热的特殊机制,指出了现有研究中存在的问题。

受制于 LM 与 S-CO<sub>2</sub> 工质本身物理性质的特殊分布,耦合换热受两侧工质浮力效应、加速效应、累积效应相互影响显著,工质的输运规律与热传递现象异常复杂,运行条件的巨大差异使耦合换热器面临大温差、高变热流等极端边界,流动将呈现多尺度、非均匀、非平衡湍流特征。针对 LM-S-CO<sub>2</sub> 与液态金属的相关研究已开展大量工作,但仍缺乏普适性计算方法及预测模型。

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